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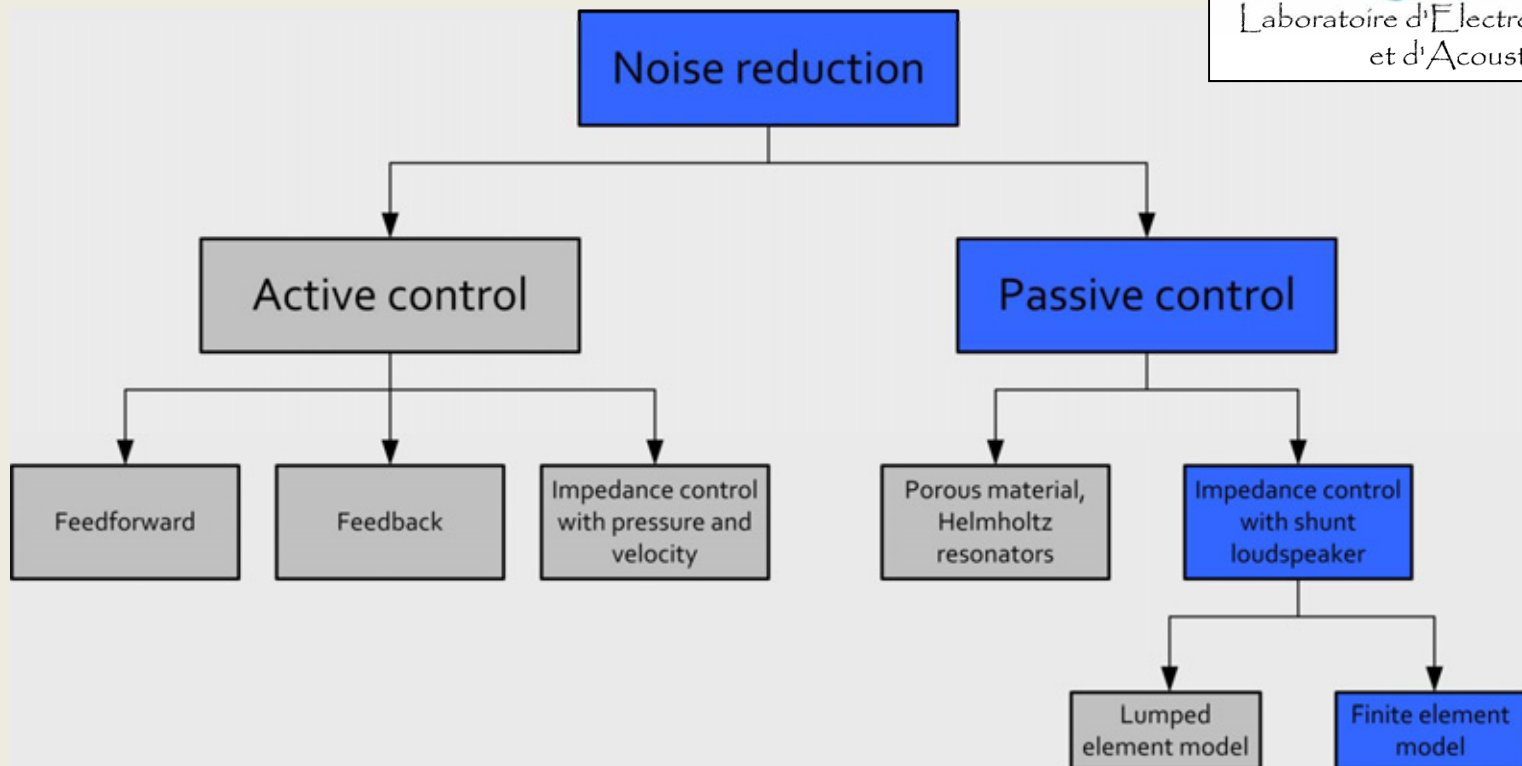
# Study of an Electroacoustic Absorber

*Anne-Sophie Moreau, Hervé Lissek and Romain Boulandet  
Ecole Polytechnique Fédérale de Lausanne, Switzerland*



# Introduction

*Laboratory of Electromagnetism and Acoustics*



# Layout of the presentation

- System dynamics modeling
  - Electroacoustic absorber dynamics
  - Acoustic waveguide
- Numerical model
  - Loudspeaker structural properties
  - Acoustic performances assessment
- Results
  - Loudspeaker characteristics
  - Acoustic performances
- Conclusion

# System dynamics modeling

System dynamics modeling

Numerical model

Results

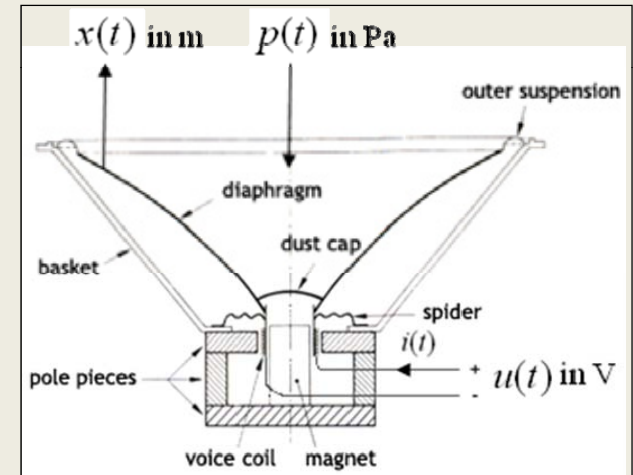
## ■ Electroacoustic absorber dynamics

- Electrical part
  - $R_e$  ( $\Omega$ ), electrical resistance
  - $L_e$  (H), electrical inductance

$Z_e$   
blocked electrical impedance
- Mechanical part
  - $R_{ms}$  ( $\text{N}\cdot\text{s}\cdot\text{m}^{-1}$ ), mechanical resistance
  - $M_{ms}$  (kg), moving mass
  - $C_{ms}$  ( $\text{m}\cdot\text{N}^{-1}$ ), mechanical compliance

$Z_m$   
free mechanical impedance
- Acoustic part
  - $M_{ar}$  ( $\text{kg}\cdot\text{m}^{-4}$ ), acoustic mass of radiation
  - $R_{ar}$  ( $\Omega\cdot\text{m}^{-4}$ ), acoustic resistance

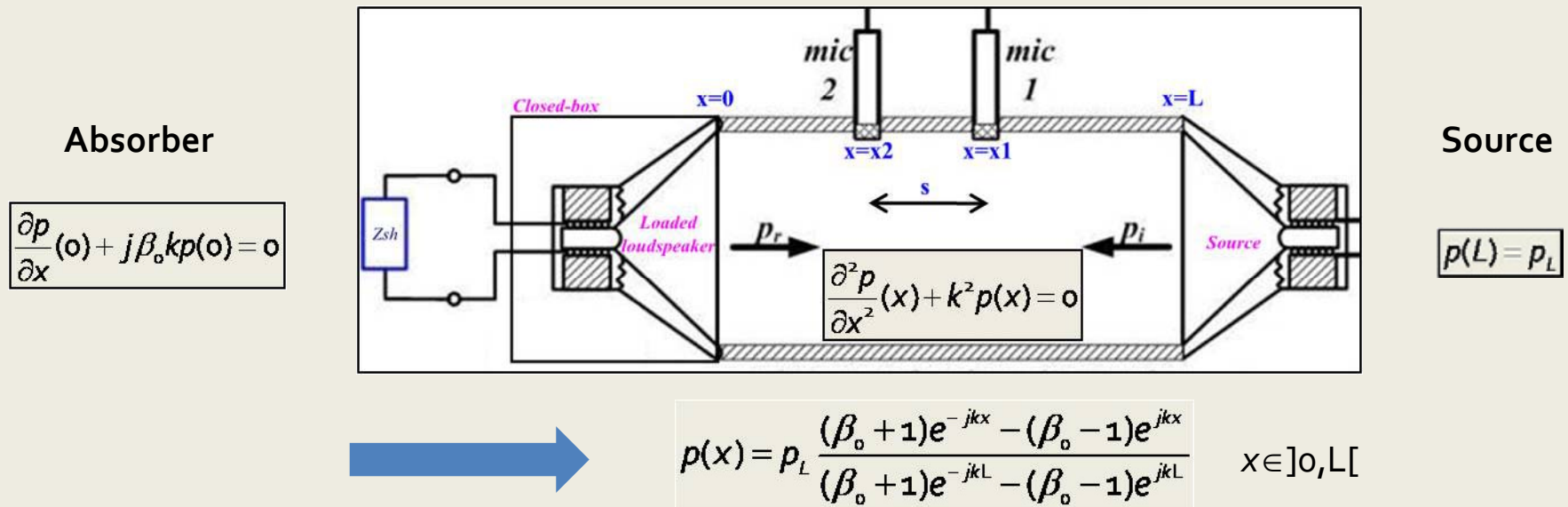
$Z_{ar}$   
acoustic impedance
- + Closed-box environment }  $Z_{ab}$  rear acoustic impedance
- + Shunt resistance }  $Z_{sh}$  shunt electrical impedance
- Coupling factors
  - $S_d$  ( $\text{m}^2$ ), diaphragm area
  - $Bl$  ( $\text{N}\cdot\text{A}^{-1}$ ), force factor



### Absorber characteristic equations

$$\begin{cases} (Bl)\underline{i} - S_d \underline{p} = (\underline{Z}_m + S_d^2 \underline{Z}_{ab}) \underline{v} \\ 0 = \underline{Z}_e \underline{i} + (Bl) \underline{v} + \underline{Z}_{sh} \underline{i} \end{cases}$$

## ■ Acoustic waveguide



- Positions of the microphones :  $s$  and  $x_1$   $\Rightarrow \Delta f_{measure} = [38 \text{ Hz}; 343 \text{ Hz}]$

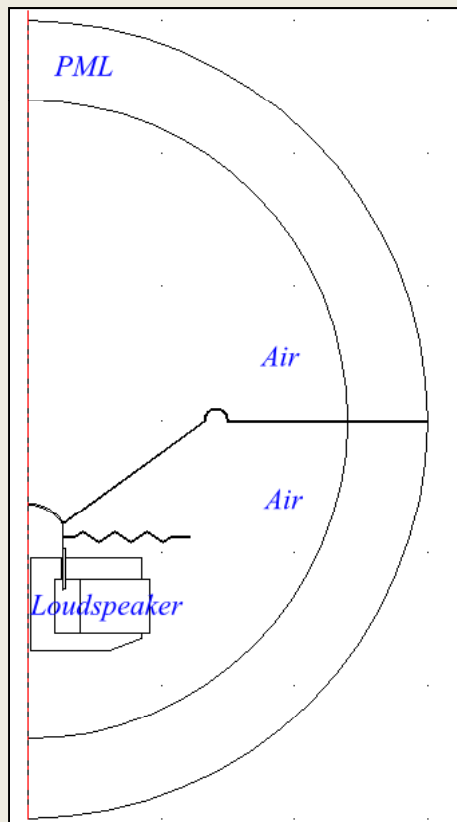
- Hypothesis :

- Plane wave if  $f < 1326 \text{ Hz}$
- Negligible attenuation because  $\alpha = 0.0075$  at 400 Hz

(ISO 10534-2 standard)

# Numerical model

## ■ Loudspeaker structural properties



- Geometric parameters
  - $a$  (m) diaphragm's radius

→  $S_d = \pi a^2$  mechano-acoustic coupling factor
- Electromagnetic parameters
  - $B_o$  (T) remanent flux density in magnet
  - $N$  number of turns in coil
  - $R_e$  ( $\Omega$ ) electrical resistance

→  $Bl = N B_o 2\pi r_{coil}$  electro-mechanical coupling factor
- Mechanical parameters
  - $E$  (Pa) Young's modulus
  - $\nu$  Poisson's ratio
  - $\rho$  ( $\text{kg}\cdot\text{m}^{-3}$ ) density
  - $\zeta$  damping

→  $|Z_{hp}(\omega)|$  ( $\Omega$ ) and  $f_s$  (Hz)

## ■ Acoustic performances assessment

### ■ Electroacoustic absorber

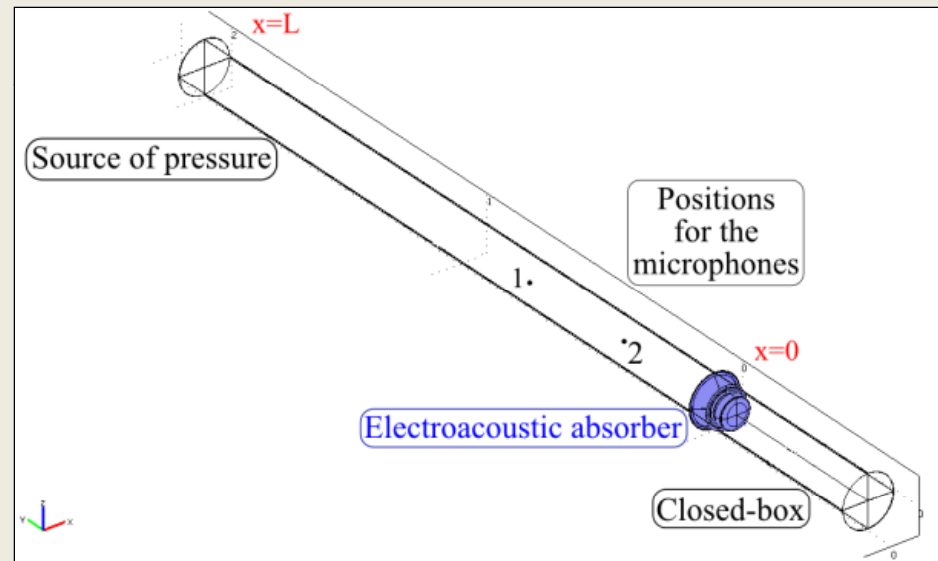
$$\begin{cases} I = \frac{V_{sh} - Blv}{Z_e} \\ V_{sh} = \frac{R_{sh}}{R_{sh} + Z_e} Blv \end{cases}$$

### ■ Source of pressure

$$p_L = 1 \text{ Pa}$$

### ■ Impedance tube and closed-box

- Sound hard walls
- Filled with air ( $\rho_o = 1.25 \text{ kg.m}^{-3}$  and  $c_o = 343 \text{ m.s}^{-1}$ )



### ■ ISO 10534-2 standard :

Pressure of microphones 1 and 2



$H_{12}$



$$r = \frac{H_{12} - e^{-iks}}{e^{iks} - H_{12}} e^{2ikx_1}$$



$$\begin{cases} \alpha = 1 - |r|^2 \\ Z_s \end{cases}$$

# Results

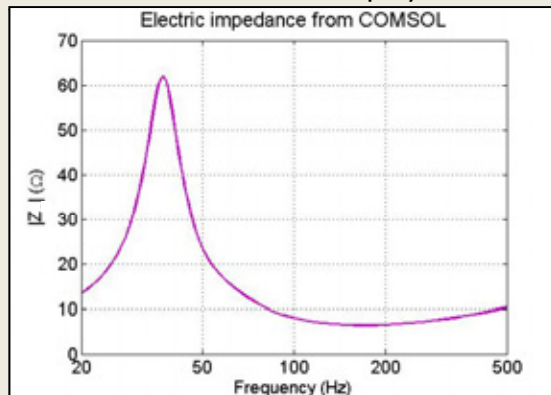
System dynamics modeling

Numerical model

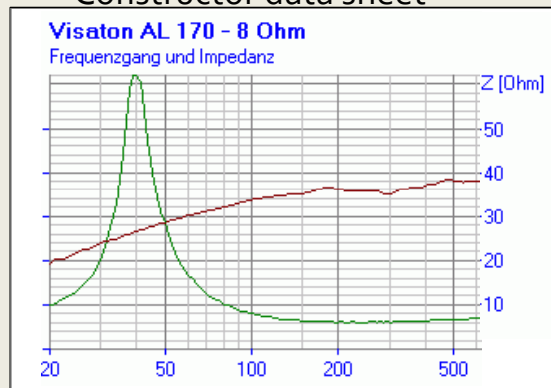
Results

## ■ Loudspeaker characteristics

- Electrical impedance  $|Z_{hp}|$ 
  - From Comsol Multiphysics®



- Constructor data sheet



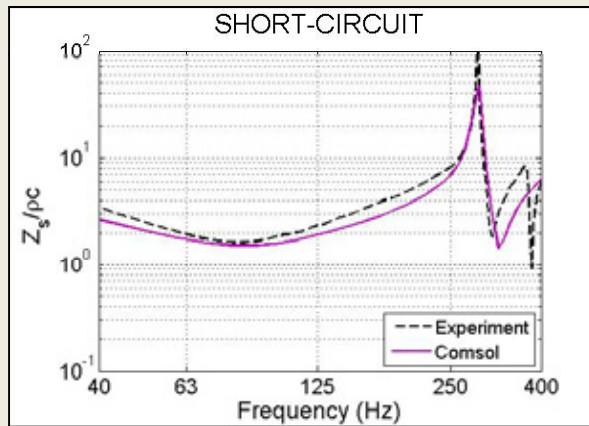
- Thiele and Small parameters

Symbol	Data sheet	Model	Error (%)
$a$ (cm)	7.4	7.5	1
$S_d$ (cm <sup>2</sup> )	133	137	3
$Bl$ (Tm)	6.9	6.7	3
$f_s$ (Hz)	38	37	3
$R_e$ (W)	5.6	5.6	0
$L_e$ (mH)	0.9	3.2	72
$R_{ms}$ (Ns.m <sup>-1</sup> )	0.8	0.78	3
$M_{ms}$ (kg)	13	12.1	7
$C_{ms}$ (mm.N <sup>-1</sup> )	1.35	1.4	4
$V_{as}$ (L)	34	39	13
$Q_{ms}$	3.88	3.9	1
$Q_{es}$	0.43	0.38	13
$Q_{ts}$	0.39	0.35	11

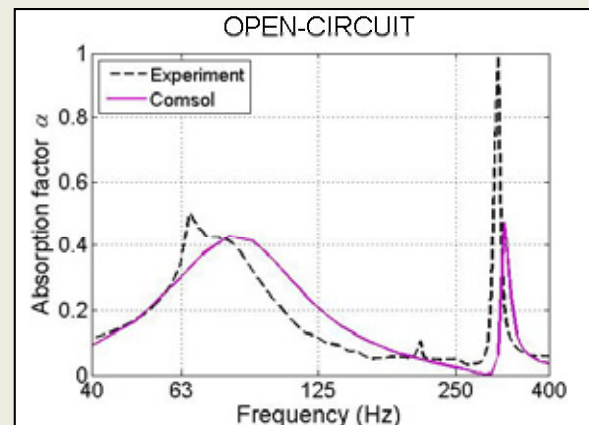
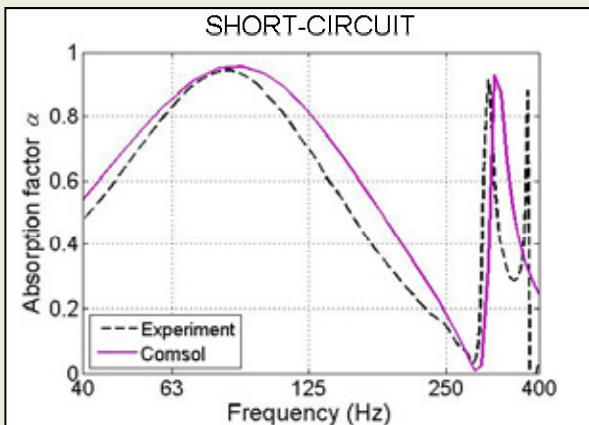
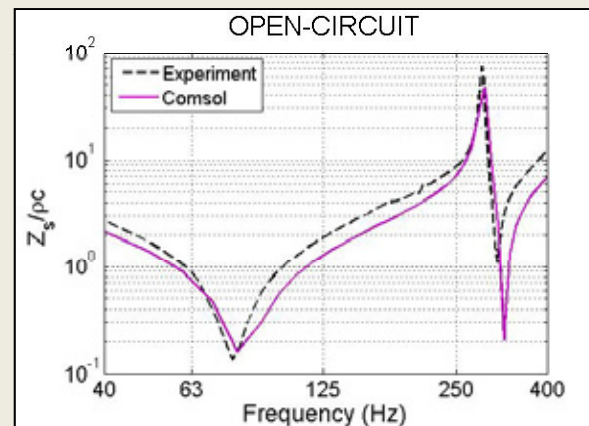


## ■ Acoustic performances (Measurement vs. Comsol)

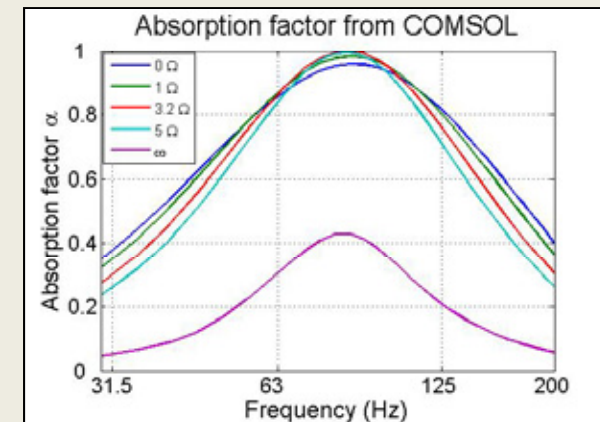
$R = 0$



$R = \infty$



$$R_{sh,opt} = \frac{(Bl)^2}{(\rho c S_d - R_{ms})} - R_e = 3.2 \Omega$$



# Conclusion

- Enhancement prospect
  - Fittings of some parameters → Cancellation of the shift in frequency
  - Optimization of the parameterization of the software
    - Perfectly Matched Layers } → Improvement of the calculation time
    - Mesh }
- Good agreement between numerical results and experiments
  - Validation of the finite element model
- Use for improving the design of electroacoustic absorber

# Thank you for your attention



*Corresponding author : [herve.lissek@epfl.ch](mailto:herve.lissek@epfl.ch)*