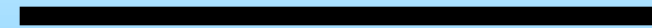


# Dynamics of Slender Structures

(Beam Theory with COMSOL PDE Solver)

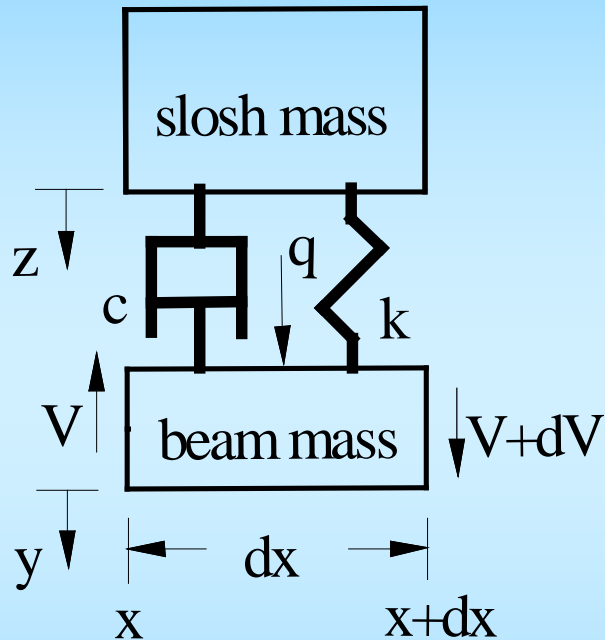
by

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## Problem Definition



$$dV + qdx = (\rho A dx) \frac{d^2 y}{dt^2}$$

$$\frac{\partial^2 M}{\partial x^2} \equiv \frac{\partial V}{\partial x} = -q + \rho A \frac{\partial^2 y}{\partial t^2}$$

$$M = -EI \frac{\partial^2 y}{\partial x^2}$$

$$\frac{\partial^2}{\partial x^2} \left( EI \frac{\partial^2 y}{\partial x^2} \right) = q - \rho A \frac{\partial^2 y}{\partial t^2}$$

# COMSOL Implementation

## Coefficient Form for PDE Models

$$e_a \frac{\partial^2 u}{\partial t^2} + d_a \frac{\partial u}{\partial t} - c_a \frac{\partial^2 u}{\partial x^2} + au = f_a$$

## Harmonic Solutions and Eigenvalue Solver

$$\partial^2 u / \partial t^2 = \lambda^2 u \qquad \partial u / \partial t = -\lambda u$$

$$-c_a \frac{\partial^2 u}{\partial x^2} + au = d_a \lambda u - e_a \lambda^2 u$$

$$\frac{\partial^2 M}{\partial x^2} = \rho A \lambda^2 y$$

$$EI \frac{\partial^2 y}{\partial x^2} + M = 0$$

## Scaled Equations

$$\xi = x/L, \eta = y/L, m = ML/EI, \tau = t/t_c,$$

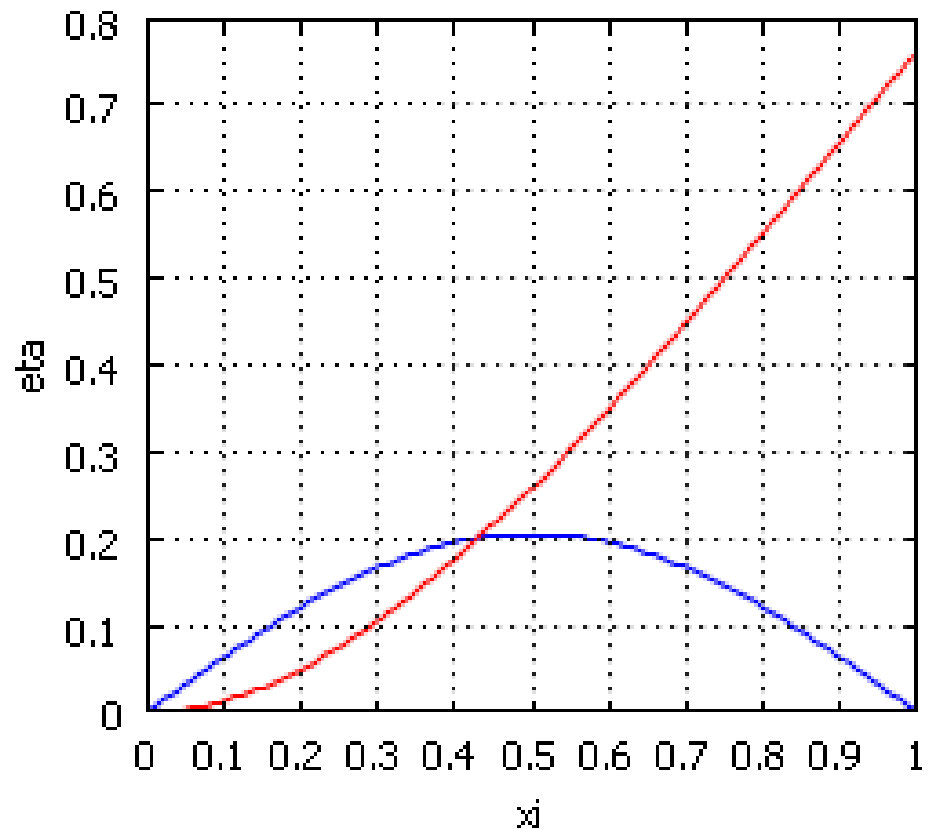
$$t_c = \sqrt{\frac{\rho AL^4}{EI}} = L^2 \sqrt{\frac{\rho A}{EI}}$$

$$\frac{\partial^2 m}{\partial \xi^2} + \frac{\partial^2 \eta}{\partial \tau^2} = 0$$

$$\frac{\partial^2 \eta}{\partial \xi^2} + m = 0$$

$$\lambda = -9.8696i = -\pi^2 i$$

$$\text{Cantilever: } \lambda = -3.515i$$



# Typical Heat Exchanger with Tubes

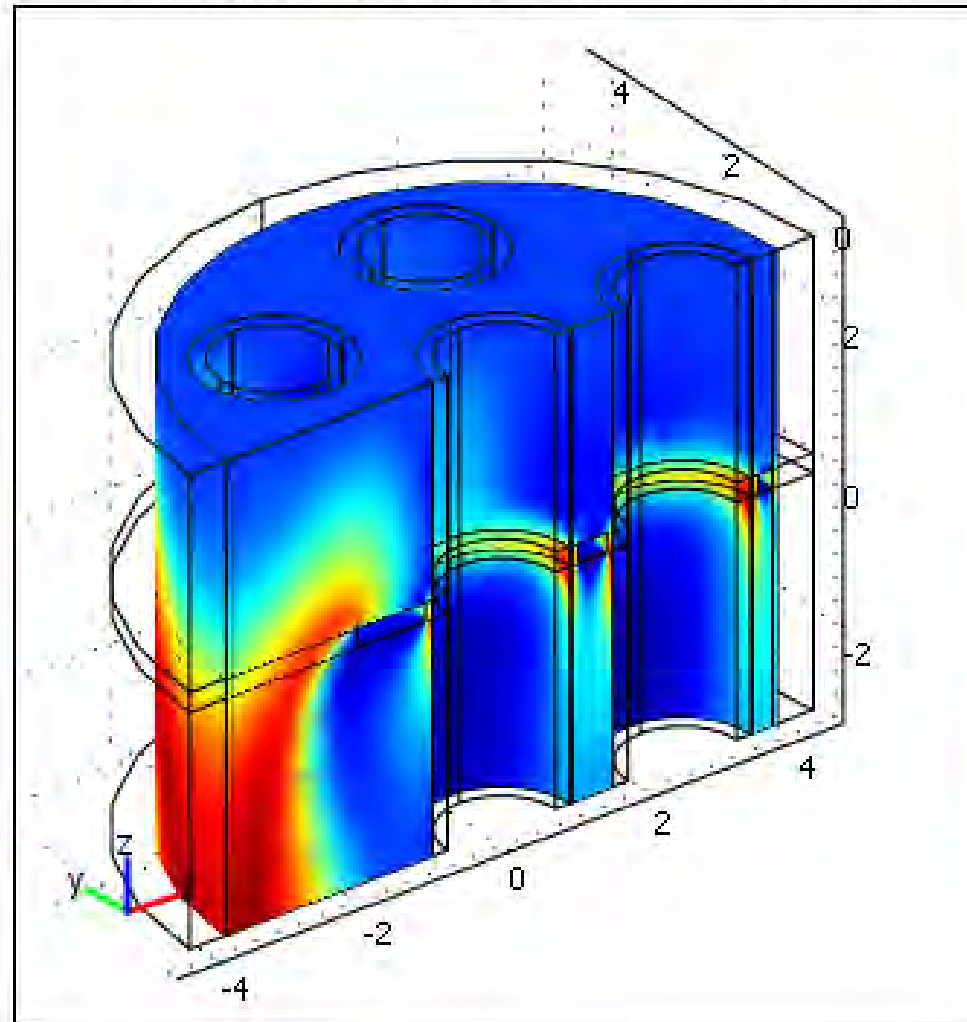
Arrow: Velocity field Streamline: Velocity field



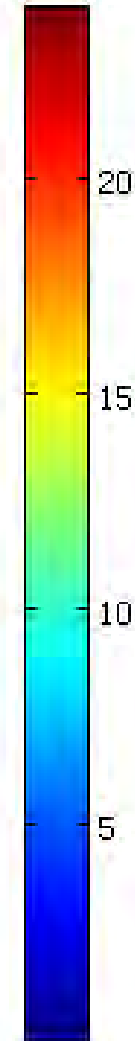
- \* Flow Induced Vibration
- \* Fluid Elastic Instability

# Heat Exchanger Flow at Baffle

Boundary: Velocity field [m/s] Streamline: Velocity field



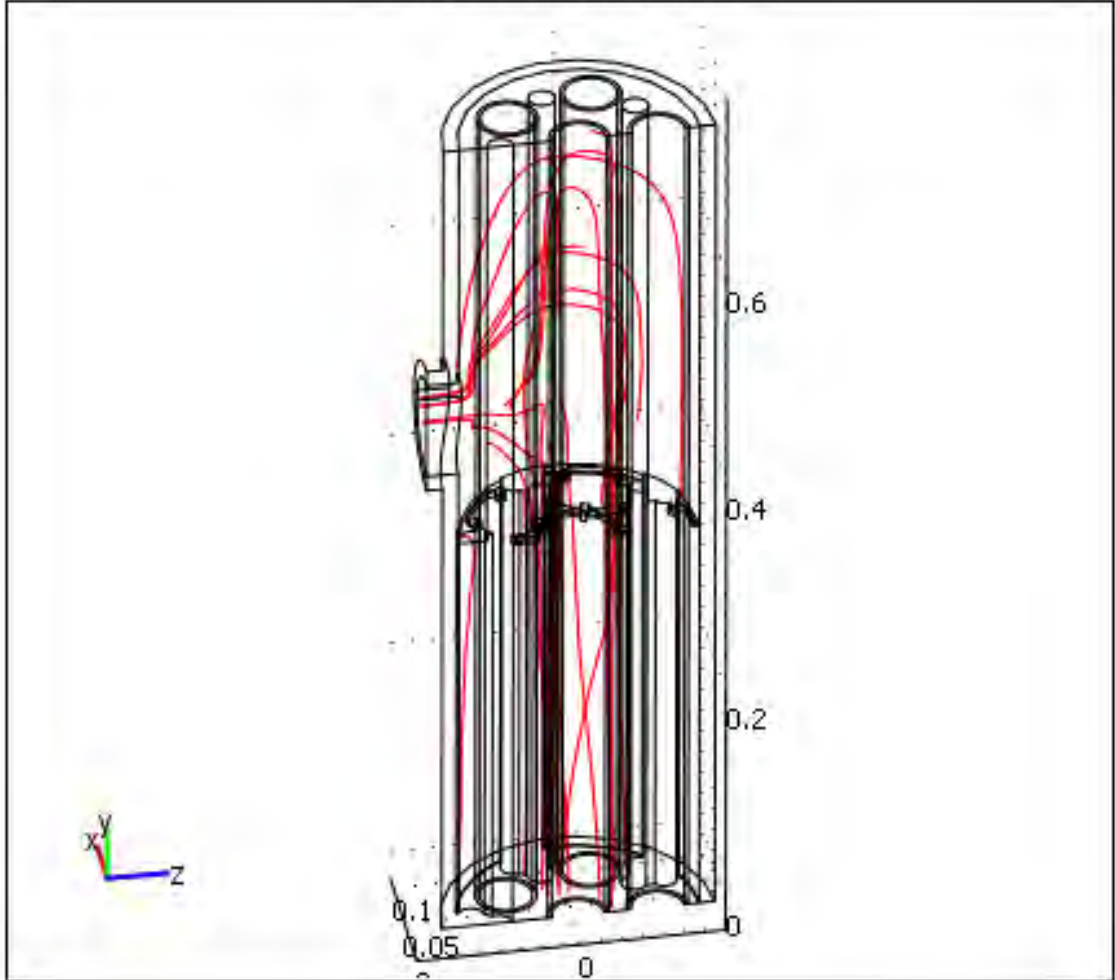
Max: 23.919



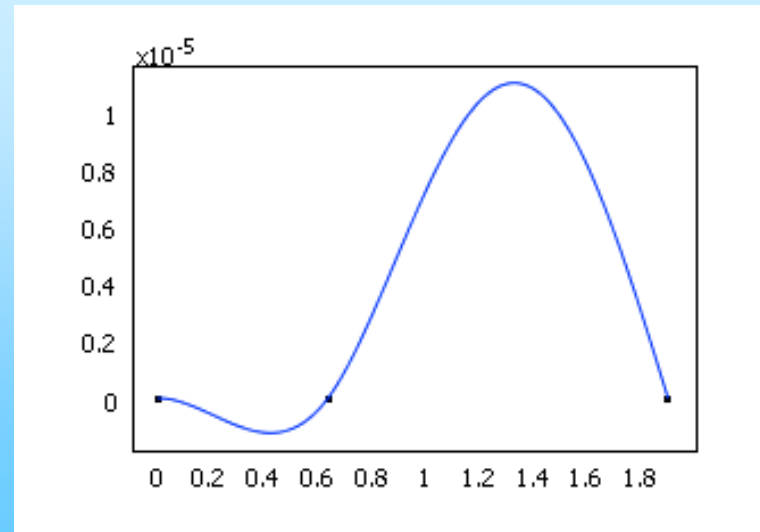
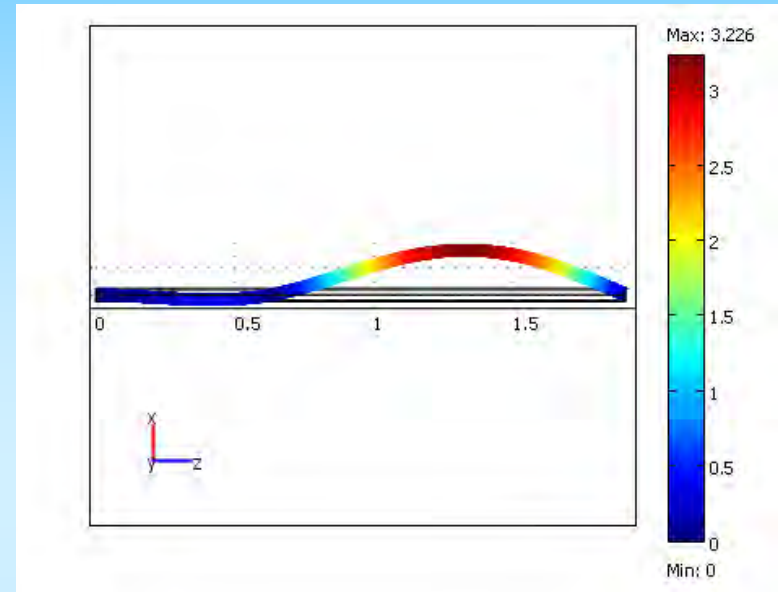
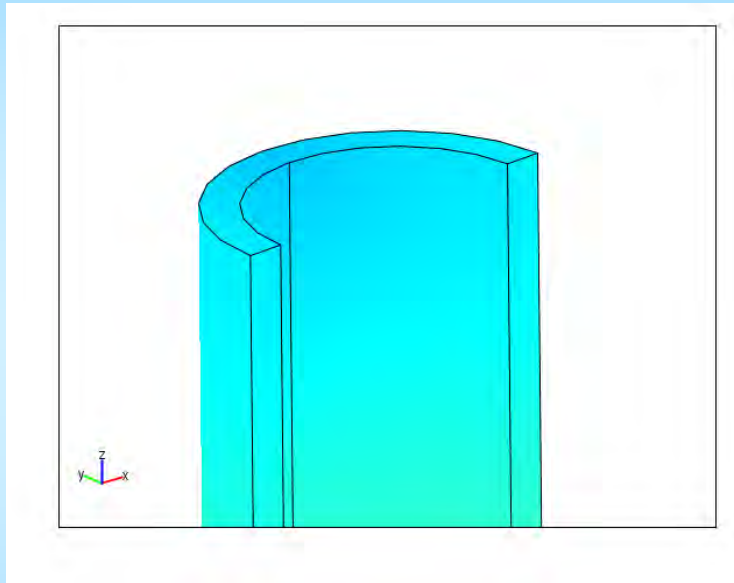
Min: 9.008e-43

# Heat Exchanger with Inlet Cross-Flow

Streamline: Velocity field



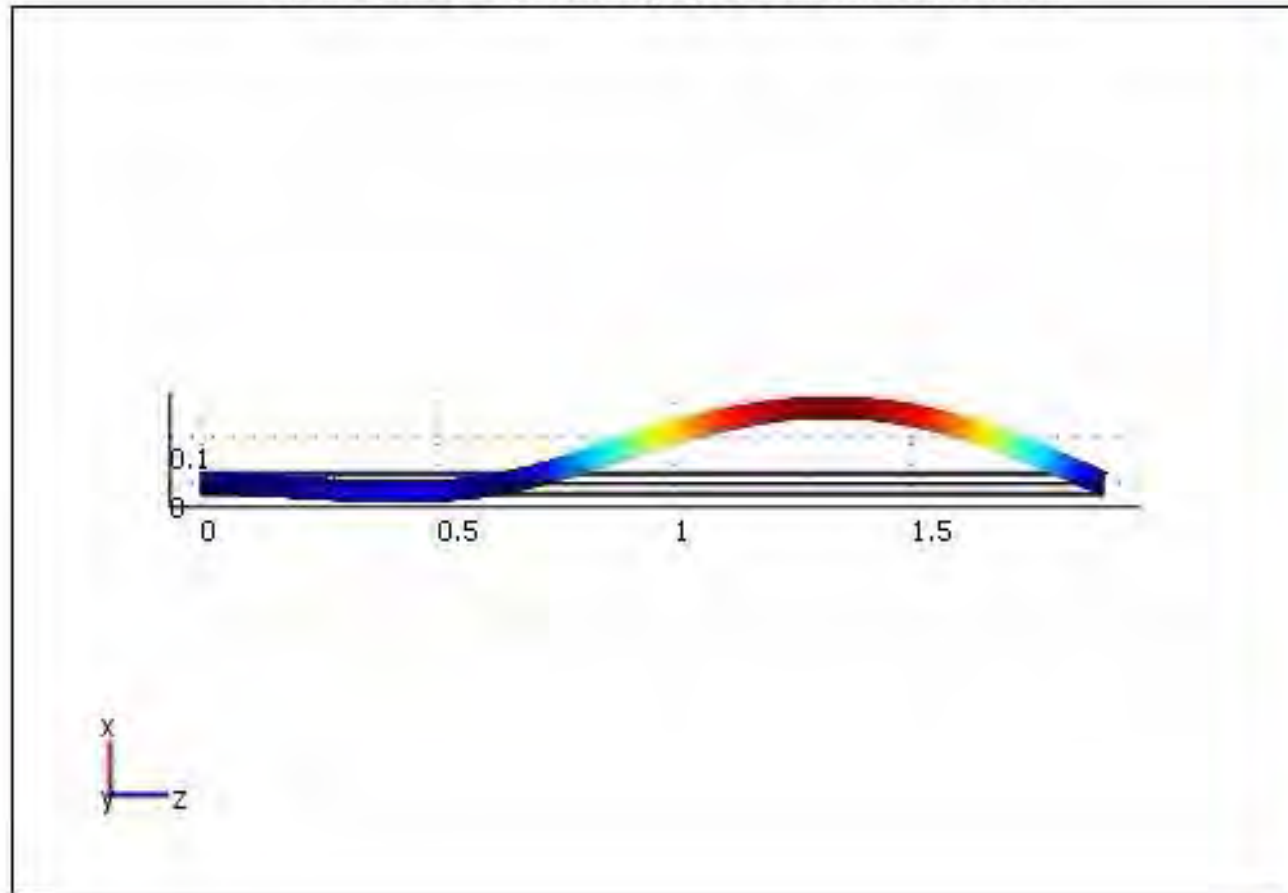
# Model of Heat Exchanger Tube





$\lambda(1) = 0.565.638844i$

Boundary: Total displacement [m] Deformation: Displacement



Max: 3.226

3

2.5

2

1.5

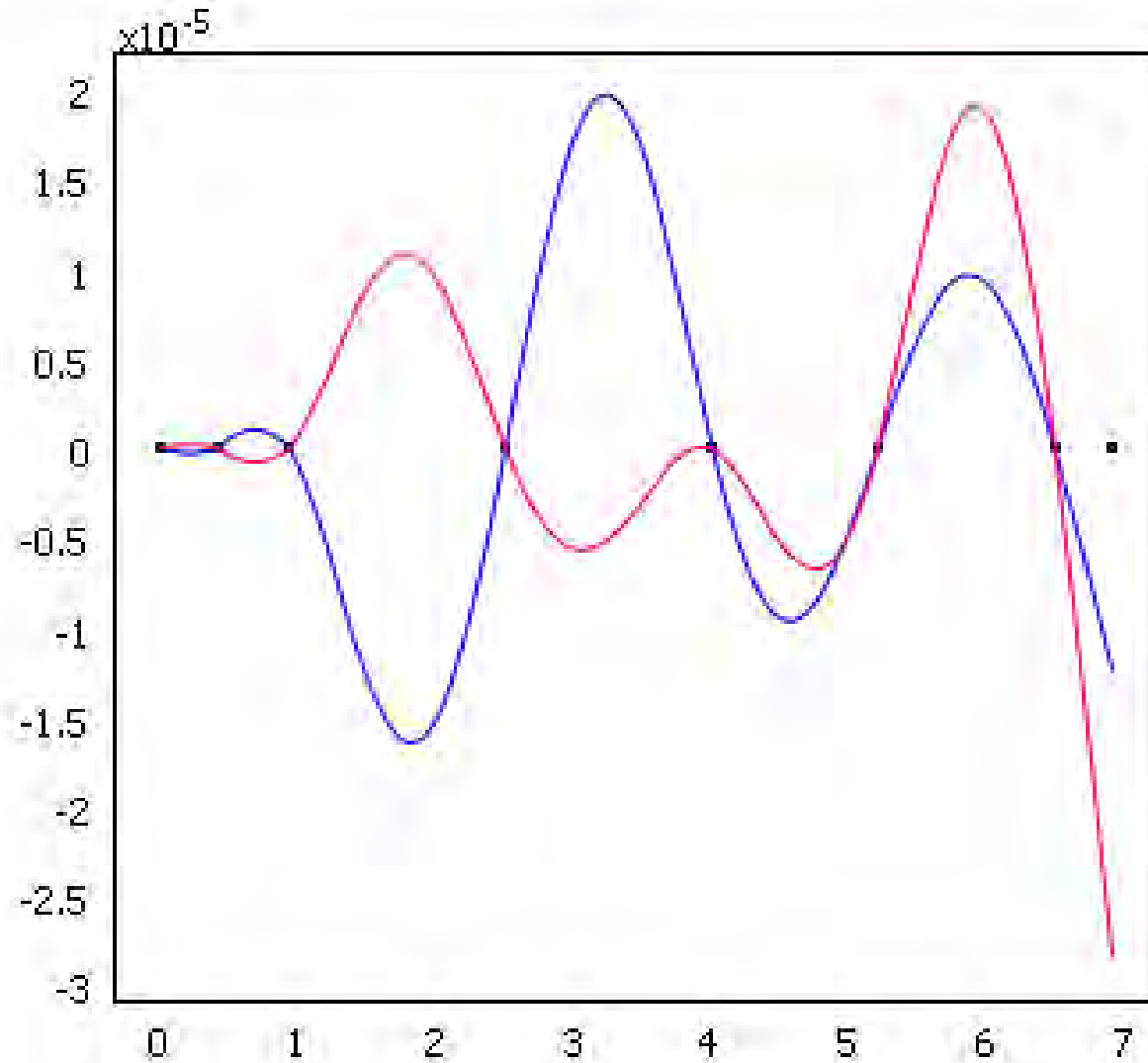
1

0.5

0

Min: 0

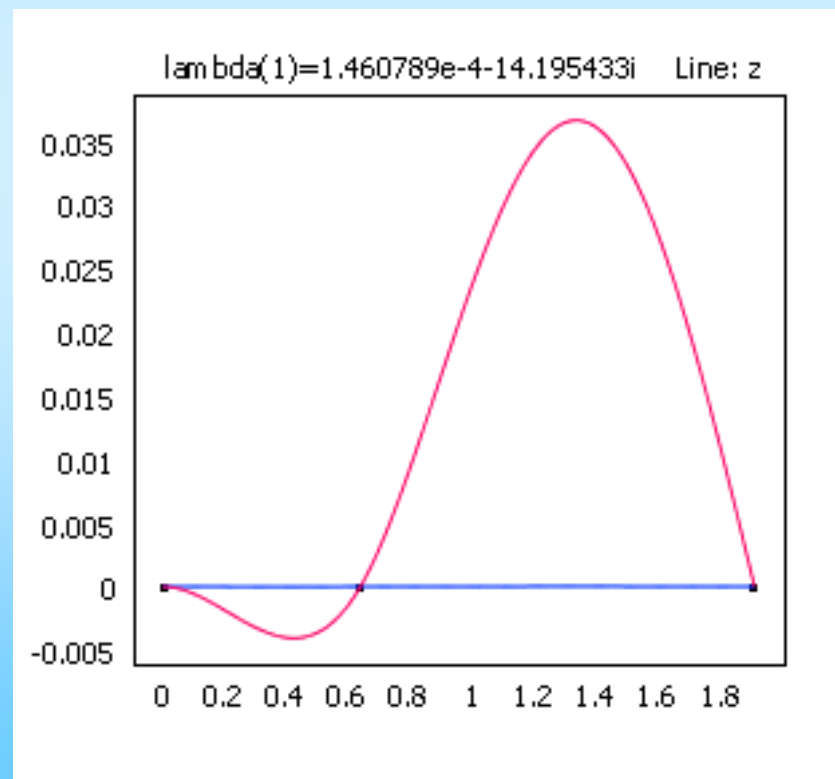
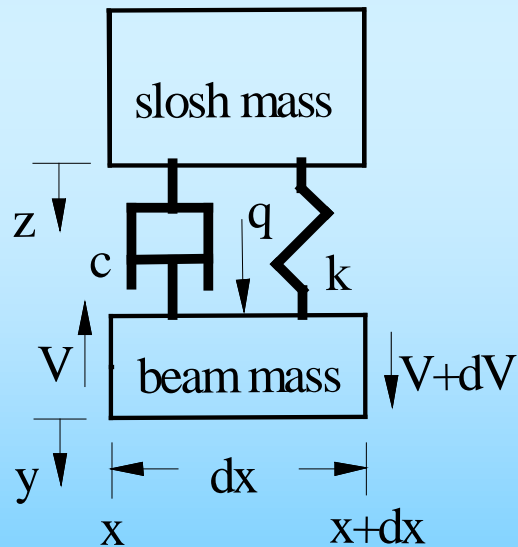
## Continuous Beam Vibration Modes



# Effect of Sloshing

$$-[c_a] \frac{\partial^2 \vec{u}}{\partial x^2} + [a] \vec{u} = [d_a] \lambda \vec{u} - [e_a] \lambda^2 \vec{u}$$

$$-\begin{bmatrix} 0 & -1 & 0 \\ -EI & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{Bmatrix} y'' \\ M'' \\ z'' \end{Bmatrix} + \begin{bmatrix} -k_s & 0 & k_s \\ 0 & 1 & 0 \\ k_s & 0 & -k_s \end{bmatrix} \begin{Bmatrix} y \\ M \\ z \end{Bmatrix} = -\begin{bmatrix} -c_s & 0 & c_s \\ 0 & 0 & 0 \\ c_s & 0 & -c_s \end{bmatrix} \begin{Bmatrix} \dot{y} \\ \dot{M} \\ \dot{z} \end{Bmatrix} - \begin{bmatrix} -\rho A & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & -m_s \end{bmatrix} \begin{Bmatrix} \ddot{y} \\ \ddot{M} \\ \ddot{z} \end{Bmatrix}$$





**The End**